

**IN THE CLAIMS:**

The text of all pending claims, (including withdrawn claims) is set forth below. Cancelled and not entered claims are indicated with claim number and status only. The claims as listed below show added text with underlining and deleted text with ~~strikethrough~~. The status of each claim is indicated with one of (original), (currently amended), (cancelled), (withdrawn), (new), (previously presented), or (not entered).

Please CANCEL claim 19 and AMEND claims 10 and 21 in accordance with the following:

1. (Previously Presented) A double-row self-aligning roller bearing, which is used to support a main shaft of a wind power generator having a blade rotor mounted on the main shaft, comprising:

an inner race;

an outer race;

left and right rows of rollers arranged between the inner race and the outer race, in which left and right bearing portions accommodating the left and right roller rows have respective load carrying capacities different from each other,

wherein one of the bearing portions that is arranged remote from the blade rotor has a load carrying capacity higher than that of the other of the bearing portions that is arranged close to the blade rotor, and each of the rollers of the row in the bearing portion arranged remote from the blade rotor has a length greater than that of each of the rollers of the row in the bearing portion arranged close to the blade rotor.

2. (Previously Presented) The double-row self-aligning roller bearing as claimed in claim 1, wherein the rollers of the left row have at least one of the dimensions and shape that is different from that of the rollers of the right row.

3. (Cancelled)

4. (Previously Presented) The double-row self-aligning roller bearing as claimed in claim 1, wherein one of the left and right roller rows is comprised of axially hollowed rollers each having an axially extending hollow.

5. (Previously Presented) The double-row self-aligning roller bearing as claimed in claim 1, wherein the left and right bearing portions have respective contact angles different from each other.

6. (Cancelled)

7. (Previously Presented) The double-row self-aligning roller bearing as claimed in claim 1, wherein the outer race is divided into two axially juxtaposed split outer races.

8. (Previously Presented) The double-row self-aligning roller bearing as claimed in claim 7, wherein a gap is provided between the two split outer races and a preload is applied to those split outer races.

9. (Cancelled)

10. (Currently Amended) A main shaft support structure for a wind power generator, comprising:

~~a main shaft having a blade rotor mounted thereon for rotation together therewith; and  
one or a plurality of bearings as defined in claim 1, disposed inside a housing to support the main shaft, in which said one or plurality of the bearings are employed in the form of a double-row self-aligning roller bearing as defined in claim 1 and include a first bearing portion arranged remote from the blade rotor and a second bearing portion arranged close to the blade rotor, and in which the first bearing portion has a load carrying capacity higher than that of the second bearing portion.~~

11. (Previously Presented) A double-row self-aligning roller bearing, which is used to support a main shaft of a wind power generator having a blade rotor mounted on the main shaft, comprising:

an inner race, an outer race and double rows of rollers arranged between the inner race and the outer race,

the bearing being in its entirety being divided into left and right split bearing portions each comprising a split inner race, a split outer race and a single row of rollers, in which elements associated with a load or a life are differentiated between the left and right split bearing portions,

wherein one of the bearing portions that is arranged remote from the blade rotor has a load carrying capacity higher than that of the other of the bearing portions that is arranged close to the blade rotor, and each of the rollers of the row in the bearing portion arranged remote from the blade rotor has a length greater than that of each of the rollers of the row in the bearing portion arranged close to the blade rotor.

12. (Previously Presented) The double-row self-aligning roller bearing as claimed in claim 11, wherein one of the elements associated with the load or the life, which are to be differentiated between the left and right split bearing portions, is a material used to form at least one of the split inner race, the split outer race and the rollers.

13. (Previously Presented) The double-row self-aligning roller bearing as claimed in claim 11, wherein one of the elements associated with the load or the life, which are to be differentiated between the left and right split bearing portions, is a surface reforming treatment to be applied to a raceway surface of at least one of the split inner race and the split outer race or a rolling surface of the rollers.

14. (Previously Presented) The double-row self-aligning roller bearing as claimed in claim 11, wherein one of the elements associated with the load or the life, which are to be differentiated between the left and right split bearing portions, is a surface roughness of a raceway surface of at least one of the split inner race and the split outer race or a rolling surface of the rollers.

15. (Cancelled)

16. (Previously Presented) The double-row self-aligning roller bearing as claimed in claim 11, wherein one of the elements associated with the load or the life, which are to be differentiated between the left and right split bearing portions, is an axial dimension of the split inner race, the split outer race and the rollers.

17. (Previously Presented) The double-row self-aligning roller bearing as claimed in claim 11, in which a gap is provided between the two split outer races and a preload is applied to those split outer races.

18-20. (Cancelled)

21. (Currently Amended) A main shaft support structure for a wind power generator, comprising:

~~a main shaft having a blade rotor mounted thereon for rotation together therewith, and one or a plurality of bearings as defined in claim 11, disposed inside a housing, in which said one or plurality of the bearings are employed in the form of a double row self-aligning roller bearing as defined in claim 11 and include a first split bearing portion arranged remote from the~~

~~blade rotor and a second split bearing portion arranged close to the blade rotor, and in which the first split bearing portion has a load carrying capacity higher or rated life longer than that of the second split bearing portion.~~

22. (Previously Presented) The double-row self-aligning roller bearing as claimed in claim 1, in which assuming that the length of the rollers of the roller row arranged close to the blade rotor is expressed by  $L1$ , the length of the rollers of the roller row arranged remote from the blade rotor is expressed by  $L2$ , and the length of a major axis of a contact ellipse generated in a plane of contact between the rollers of one of the roller rows and the inner race is expressed by  $A$ , the following dimensional relationship establishes:

$$L1 < L2 \text{ and } L1 > A.$$

23. (Previously Presented) The double-row self-aligning roller bearing as claimed in claim 22, in which the rollers are symmetrical rollers having a maximum diameter positioned at a location intermediate of the length of the rollers.

24. (Previously Presented) The double-row self-aligning roller bearing as claimed in claim 22, in which the rollers are asymmetrical rollers having a maximum diameter positioned at a location displaced from a point intermediate of the length of the rollers.

25. (Previously Presented) The double-row self-aligning roller bearing as claimed in claim 1, in which assuming that the radius of curvature of a ridge of the rollers of one of the roller rows is expressed by  $R1$ , the radius of curvature of a ridge of the rollers of the other of the roller rows is expressed by  $R2$ , the radius of curvature of the raceway surface of the inner race, with which the rollers of one of the roller rows contact, is expressed by  $N1$ , and the radius of curvature of the raceway surface of the inner race, with which the rollers of the other of the roller rows contact, is expressed by  $N2$ , the following dimensional relationship establishes:

$$N1/R1 < N2/R2.$$

26. (Previously Presented) The double-row self-aligning roller bearing as claimed in claim 25, in which the radius of curvature  $R1$  is greater than the radius of curvature  $R2$ .

27. (Previously Presented) The double-row self-aligning roller bearing as claimed in claim 25, in which the radius of curvature  $N1$  is smaller than the radius of curvature  $N2$ .

28. (Previously Presented) The double-row self-aligning roller bearing as claimed in claim

25, in which one of the rows of the rollers having the radius of curvature  $R_1$  have a length smaller than that of the other of the rows of the rollers having the radius of curvature  $R_2$ .